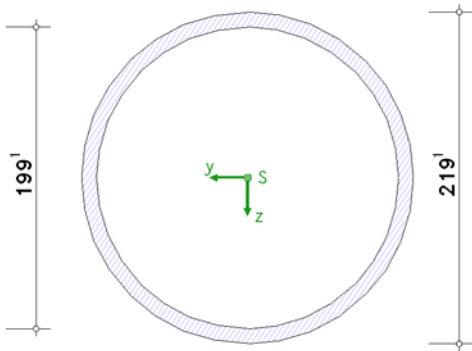


POS. 27: KINDMANN/KRÜGER 2.9.7

verification of cross-section EC 3-1-8 (12.10), NA: Deutschland

1. input report

scale 1:5.0



steel

steel grade S235

material safety factor

resistance of cross-sections $\gamma_{M0} = 1.00$

geometry

section R 219.1 x 10.0(w)

resistance

plastic verification acc. to EC 3-1-1, 6.2

internal forces and moments referring to local axes of cross-section

σ -generating forces (N, M) work at centroid, τ -generating forces (V, T_t) work at shear center

Lk 1: $V_{z,Ed} = 150.00$ kN, $V_{y,Ed} = 60.00$ kN, $T_{t,Ed} = 14.00$ kNm

Lk 2: $N_{Ed} = 240.00$ kN, $M_{y,Ed} = 85.00$ kNm, $V_{z,Ed} = 150.00$ kN, $M_{z,Ed} = 25.00$ kNm, $V_{y,Ed} = 60.00$ kN, $T_{t,Ed} = 14.00$ kNm

notes

buckling is not investigated.

2. Lk 1

2.1. verification of cross-section

internal forces and moments: $V_z = 150.00$ kN, $V_y = 60.00$ kN, $T_t = 14.00$ kNm

2.1.1. plastic verification

circular hollow section: $V = (V_y^2 + V_z^2)^{1/2} = 161.55$ kN

plastic verification for $V = 161.55$ kN, $T_t = 14.00$ kNm

plastic values: $V_{pl,Rd} = A_v \cdot f_y / (3^{1/2} \cdot \gamma_{M0}) = 567.40$ kN

torsion

$\tau_{t,Ed} = |T_{t,Ed}| / W_t = 20.38$ N/mm², $W_t = 686.80$ cm³

verification: $\tau_{t,Ed} / \tau_{Rd} = 0.150 < 1$ **ok**

shear force and torsion

deduction: $\tau_{t,Ed} = |T_{t,Ed}| / W_t = 20.38$ N/mm², $W_t = 686.80$ cm³

$f_T = 1 - \tau_{t,Ed} / \tau_{Rd} = 0.850 \Rightarrow V_{pl,T,Rd} = V_{pl,Rd} \cdot f_T = 482.15$ kNm

shear force

verification: $V_{Ed} / V_{pl,T,z,Rd} = 0.335 < 1$ **ok**

3. Lk 2

3.1. verification of cross-section

internal forces and moments: $N = 240.00$ kN, $M_y = 85.00$ kNm, $V_z = 150.00$ kN, $M_z = 25.00$ kNm, $V_y = 60.00$ kN, $T_t = 14.00$ kNm

3.1.1. plastic verification

circular hollow section: $V = (V_y^2 + V_z^2)^{1/2} = 161.55 \text{ kN}$, $M = (M_y^2 + M_z^2)^{1/2} = 88.60 \text{ kNm}$

plastic verification for $N = 240.00 \text{ kN}$, $M = 88.60 \text{ kNm}$, $V = 161.55 \text{ kN}$, $T_t = 14.00 \text{ kNm}$

plastic values: $N_{pl,Rd} = A \cdot f_y / \gamma_{M0} = 1543.73 \text{ kN}$, $M_{pl,Rd} = W_{pl} \cdot f_y / \gamma_{M0} = 100.83 \text{ kNm}$

$V_{pl,Rd} = A_v \cdot f_y / (3^{1/2} \cdot \gamma_{M0}) = 567.40 \text{ kN}$

torsion

$\tau_{t,Ed} = |T_{t,Ed}| / W_t = 20.38 \text{ N/mm}^2$, $W_t = 686.80 \text{ cm}^3$

shear force and torsion

deduction: $\tau_{t,Ed} = |T_{t,Ed}| / W_t = 20.38 \text{ N/mm}^2$, $W_t = 686.80 \text{ cm}^3$

$f_T = 1 - \tau_{t,Ed} / \tau_{Rd} = 0.850 \Rightarrow V_{pl,T,Rd} = V_{pl,Rd} \cdot f_T = 482.15 \text{ kNm}$

shear force

reduction factor: $\rho = (2 \cdot V_{Ed} / V_{pl,T,Rd} - 1)^2 = 0.109$

axial force and shear force/torsion

$N_{pl,Rd} = A \cdot (1 - \rho) \cdot f_y / \gamma_{M0} = 1375.76 \text{ kN}$

bending and shear force/torsion

$M_{pl,V,Rd} = W_{pl} \cdot (1 - \rho) \cdot f_y / \gamma_{M0} = 89.86 \text{ kNm}$

bending

verification: $|M_{y,Ed}| / M_{pl,V,y,Rd} = 0.986 < 1$ **ok**

4. final result

maximum utilization [Lk 2]: resistance max U = 0.986 < 1 **ok**

verification succeeded